

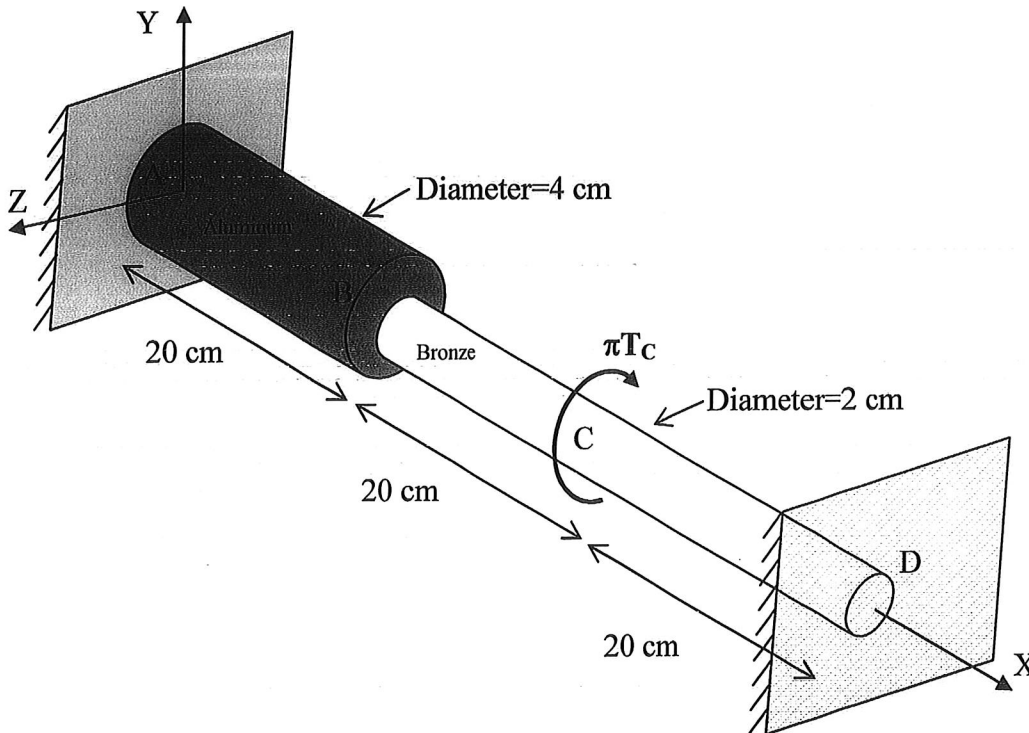
February 15, 2012

Instructor _____

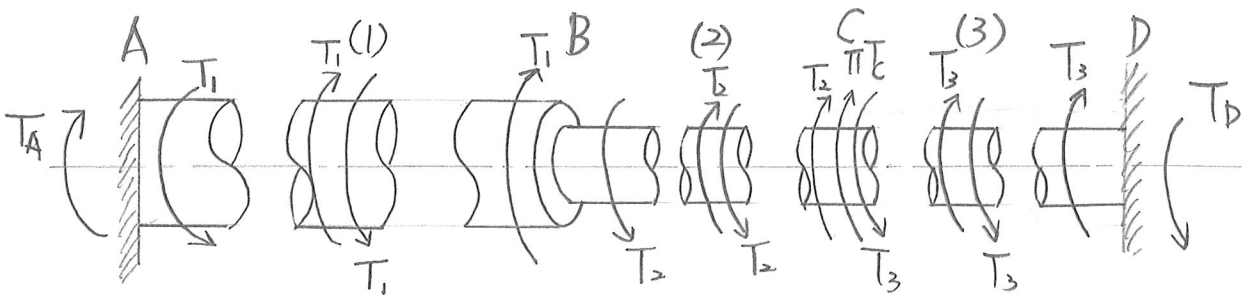
PROBLEM #2 (25 points)

The stepped rod shown below is made of Aluminum and bronze (shear modulus of aluminum $G_{al} = 30$ GPa and shear modulus of bronze $G_{br} = 45$ GPa) and is fixed to the walls at A and D. The aluminum section is 20 cm long and has a diameter of 4 cm. The bronze section is 40 cm long and has a diameter of 2 cm. An external torque πT_C is applied as shown. The shear yield stress of the aluminum and bronze are $\tau_Y = 200$ MPa and 128 MPa respectively. Determine the maximum allowable applied torque at C, T_C ?

Note: the torque applied is πT_C .



Sol)



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X

$$\tau_1 = 2 \times 10^8 \text{ [Pa]} , G_1 = 30 \times 10^9 \text{ [Pa]}$$

$$\tau_2 = 1.28 \times 10^8 \text{ [Pa]} , G_2 = 45 \times 10^9 \text{ [Pa]}$$

$$\tau_3 = 1.28 \times 10^8 \text{ [Pa]} , G_3 = 45 \times 10^9 \text{ [Pa]}$$

$$d_1 = \frac{1}{25} \text{ [m]} , L_1 = \frac{1}{5} \text{ [m]} , J_1 = \frac{\pi d_1^4}{32} = \frac{\pi}{125 \times 10^5} \text{ [m}^4\text{]}$$

$$d_2 = \frac{1}{50} \text{ [m]} , L_2 = \frac{1}{5} \text{ [m]} , J_2 = \frac{\pi d_2^4}{32} = \frac{\pi}{2 \times 10^8} \text{ [m}^4\text{]}$$

$$d_3 = \frac{1}{50} \text{ [m]} , L_3 = \frac{1}{5} \text{ [m]} , J_3 = \frac{\pi d_3^4}{32} = \frac{\pi}{2 \times 10^8} \text{ [m}^4\text{]}$$

$$i) \sum M_A = -T_A + T_1 = 0$$

$$\therefore T_A = T_1 \quad \text{--- (1)}$$

$$ii) \sum M_B = -T_1 + T_2 = 0$$

$$\therefore T_1 = T_2 \quad \text{--- (2)}$$

$$iii) \sum M_C = -\pi T_C - T_2 + T_3 = 0$$

$$\therefore T_3 = \pi T_C + T_2 \quad \text{--- (3)}$$

$$iv) \sum M_D = T_D - T_3 = 0$$

$$\therefore T_D = T_3 \quad \text{--- (4)}$$

$$v) \phi_1 = \phi_A - \phi_B$$

$$\phi_2 = \phi_B - \phi_C$$

$$\phi_3 = \phi_C - \phi_D$$

$$\therefore \phi_1 + \phi_2 + \phi_3 = 0 \quad \text{--- (5)}$$

In Eq (5),

$$\phi_2 = \left(\frac{\pi L_2}{G_2 J_2} \right) T_2 = -\phi_1 - \phi_3 = - \left[\frac{T_1 L_1}{G_1 J_1} + \frac{T_3 L_3}{G_3 J_3} \right]$$

$$\therefore T_1 = T_2 , T_1 = T_2 = - \frac{G_2 J_2}{L_2} \left[\frac{T_1 L_1}{G_1 J_1} + \frac{T_3 L_3}{G_3 J_3} \right]$$

$$\therefore T_1 = T_2 = -0.914 T_3$$

Using Eq. (3)

$$T_3 = 1.641 T_c$$

Meanwhile

$$\tau_1 = \frac{T_1 d_1}{J_1} = \frac{-0.914 \times 1.641 T_c d_1}{J_1} = 2 \times 10^8 \text{ [Pa]} \quad \therefore T_c = -1675 \text{ [N}\cdot\text{m]}$$

$$\tau_2 = \frac{T_2 d_2}{J_2} = \frac{-0.914 \times 1.641 T_c d_2}{J_2} = 1.28 \times 10^8 \text{ [Pa]} \quad \therefore T_c = -134 \text{ [N}\cdot\text{m]}$$

$$\tau_3 = \frac{T_3 d_3}{J_3} = \frac{1.641 T_c d_3}{J_3} = 1.28 \times 10^8 \text{ [Pa]} \quad \therefore T_c = 122.5 \text{ [N}\cdot\text{m]}$$

Therefore,

$$T_c = 122.5 \text{ [N}\cdot\text{m]}$$

Verification)

From the final T_c ,

$$T_1 = -183.83 \text{ [N}\cdot\text{m]}$$

$$T_2 = -183.83 \text{ [N}\cdot\text{m]}$$

$$T_3 = 201.06 \text{ [N}\cdot\text{m]}$$

$$\tau_1 = \frac{T_1 d_1}{J_1} = -1.46 \times 10^8 \text{ [Pa]}$$

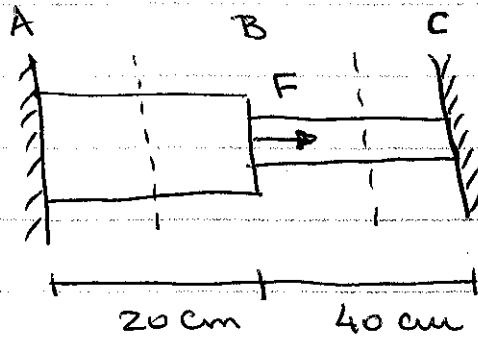
$$\tau_2 = \frac{T_2 d_2}{J_2} = -1.19 \times 10^8 \text{ [Pa]}$$

$$\tau_3 = \frac{T_3 d_3}{J_3} = 1.28 \times 10^8 \text{ [Pa]}$$

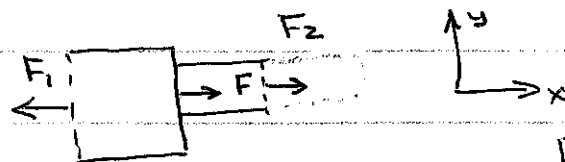
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Therefore, with $T_0 = 122.5 \text{ [N.m]}$, the shear stress in each section is safe. Any torque larger than $T_0 = 122.5 \text{ [N.m]}$ will be safe in the Aluminium section, but will fail the bronze.

Problem 1



Normal stress
in Aluminum and steel
sections.



$$\sum F_x = 0$$
$$F + F_2 - F_1 = 0 \quad (1)$$

Compatibility condition

$$e_1 + e_2 = 0 \quad (2)$$

Force - elongation

$$e_1 = \frac{F_1 L_1}{A_1 E_1} \quad (3)$$

$$e_2 = \frac{F_2 L_2}{A_2 E_2} \quad (4)$$

Solve for F_1, F_2 .

③ and ④ in ②

$$\frac{F_1 \cdot 20 \cdot 10^{-2} \text{ m}}{10 \text{ GPa} \cdot \frac{1}{4} \pi \cdot (2 \cdot 10^{-2} \text{ m})^2} + \frac{F_2 \cdot 40 \cdot 10^{-2} \text{ m}}{30 \text{ GPa} \cdot \frac{1}{3} \pi \cdot (10^{-2} \text{ m})^2}$$

$$\frac{F_1}{4} + \frac{2F_2}{3} = 0$$

$$F_1 = -\frac{8}{3}F_2 \rightarrow \text{in } ①$$

$$10 \text{ kN} + F_2 + \frac{8}{3}F_2 = 0$$

$$\frac{11}{3}F_2 = -10 \text{ kN}$$

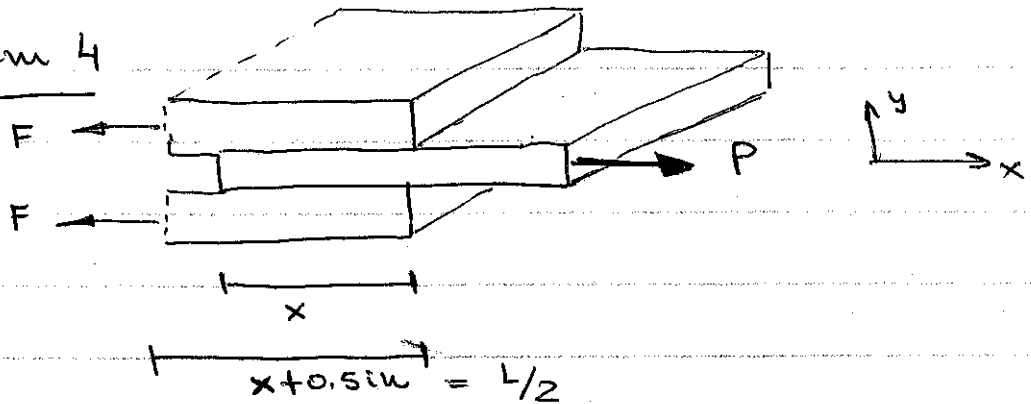
$$F_2 = -\frac{30}{11} \text{ kN}$$

$$F_1 = \frac{80}{11} \text{ kN}$$

$$\sigma_1 = F_1/A_1 = -8.68 \text{ MPa}$$

$$\sigma_2 = F_2/A_2 = 5.99 \text{ MPa}$$

Problem 4



$$\sum F_x = 0 \quad F = P/2$$

$$\tau = \frac{P/2}{\text{Area}} = \frac{P/2}{x \cdot 5 \text{ in}}$$

$$\tau < \tau_{\text{allow}} = 100 \text{ psi}$$

$$\frac{P/2}{x \cdot 5 \text{ in}} < 100 \text{ psi}$$

$$\Rightarrow x > 10 \text{ in}$$

$$\Rightarrow L = 2x + 1 \text{ in}$$

$$L = 21 \text{ in.}$$

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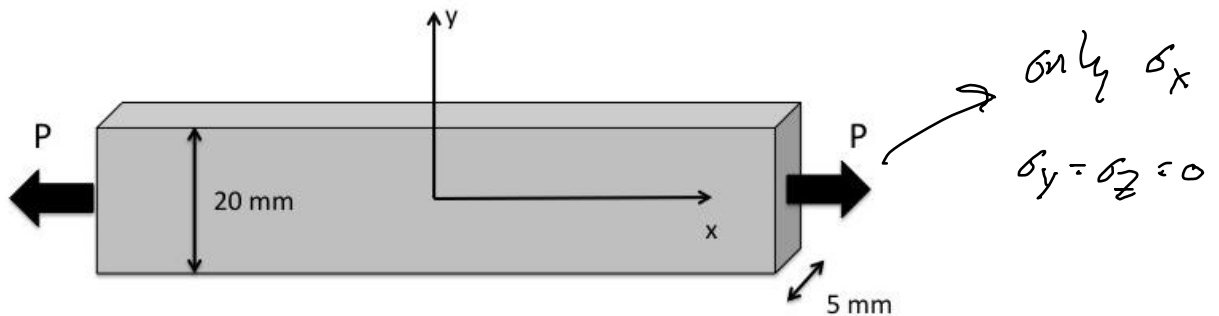
Instructor Sadeghi Koslowski

PROBLEM #3 (25 points)

A rectangular bar with a base and height of 5 mm and 20 mm respectively is subjected to a tensile load P . The longitudinal (x) and transverse (y) direction strains were measured as $\epsilon_x = 0.0015$ and $\epsilon_y = 0.0005$.

Determine:

- The Poisson's ratio of the specimen.
- If the strains were measured with an axial load of $P = 25$ kN. What is the Young's modulus of the specimen?



$$\epsilon_x = \frac{1}{E} \sigma_x \quad \epsilon_y = \frac{1}{E} (0 - \nu \sigma_x) = -\frac{\nu \sigma_x}{E} = -\nu \epsilon_x$$

$$(a) \nu = -\frac{\epsilon_y}{\epsilon_x} = \frac{+0.0005}{0.0015} = \boxed{+\frac{1}{3}}$$

$$(b) E = \frac{\sigma_x}{\epsilon_x} = \frac{P}{A \epsilon_x} = \frac{P}{w \cdot h \cdot \epsilon_x} = \frac{25 \text{ kN}}{(5 \cdot 20) \text{ mm}^2 \cdot 0.0015}$$

$$\boxed{E = 166.7 \text{ GPa}}$$