

**ME 315**  
**Exam 1 Solutions**  
**8:00 -9:00 PM**  
**Tuesday, February 10, 2009**

1. (25 points) Consider a composite slab composed of a layer of plastic bonded to a layer of copper. The thickness of the plastic layer is  $15\text{ mm}$  while that of the copper layer is  $10\text{ mm}$ . The cross-sectional area of the slab is  $100\text{ cm}^2$ . There is contact resistance at the plastic-Cu interface due to the bonding. During testing, the plastic side is subjected to a hot fluid stream at  $T_{\infty 1} = 400\text{ K}$  and a heat transfer coefficient of  $200\text{ W/m}^2\text{K}$ . The copper side is subjected to surrounding air at temperature  $T_{\infty 2} = 300\text{ K}$ , with a heat transfer coefficient of  $100\text{ W/m}^2\text{K}$ .

You are also given the following data:

Thermal contact conductance at plastic-Cu interface =  $420\text{ W/m}^2\text{K}$ .

$k_{\text{plastic}} = 2\text{ W/mK}$ ;  $k_{\text{Cu}} = 400\text{ W/mK}$

(a) Draw a thermal circuit for the composite showing all relevant thermal resistances and their numerical values.

Solution



$$L_{\text{plastic}} = t_{\text{plastic}} = 15\text{mm} = 0.015\text{m}$$

$$L_{\text{Cu}} = t_{\text{Cu}} = 10\text{mm} = 0.010\text{m}$$

$$A_c = 100\text{cm}^2 \frac{(1\text{m})^2}{(100\text{cm})^2} = 0.01\text{m}^2$$

$$\therefore R_{\text{conv},1} = \frac{1}{h_1 A} = \frac{1}{(200\text{W/m}^2\text{K})(0.01\text{m}^2)} = \boxed{0.5\text{ K/W}}$$

$$R_{\text{cond},P} = \frac{L_P}{k_P A} = \frac{0.015\text{m}}{(2\text{W/mK})(0.01\text{m}^2)} = \boxed{0.75\text{ K/W}}$$

$$R_{contact} = \frac{1}{k_{contact} A} = \frac{1}{(420 W/m^2 K)(0.01 m^2)} = \boxed{0.2381 K/W}$$

$$R_{cond, Cu} = \frac{L_{Cu}}{k_{Cu} A} = \frac{0.010 m}{(400 W/mK)(0.01 m^2)} = \boxed{0.0025 K/W}$$

$$R_{conv, 2} = \frac{1}{h_2 A} = \frac{1}{(100 W/m^2 K)(0.01 m^2)} = \boxed{0.1 K/W}$$

(b) What is the drop in temperature across the plastic-Cu interface in K (i.e. the temperature drop across the contact resistance)?

Solution

$$R_{tot} = 0.5 + 0.75 + 0.2381 + 0.0025 + 0.1$$

$$= 1.5906 K/W$$

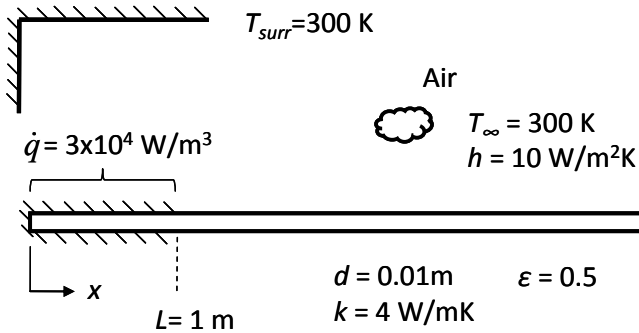
$$\Delta T_{interface} = 14.95268 K$$

$$q = \frac{T_{\infty,1} - T_{\infty,2}}{R_{tot}} = \frac{400 - 300}{1.5906} = 62.8 W$$

$$\Delta T_{interface} = R_{contact} \cdot q$$

$$= 0.2381 \cdot 62.8 = \boxed{14.95268 K}$$

2. (35 Points) A very long circular rod of  $0.01\text{m}$  diameter and thermal conductivity  $k = 4\text{ W/mK}$  is placed in a large enclosure. A small portion of the rod ( $0 \leq x \leq L$ ) is perfectly insulated and experiences uniform heat generation,  $\dot{q} = 3 \times 10^4\text{ W/m}^3$ . The rod loses heat by convection and radiation, as shown. The rod has a gray surface with emissivity  $\varepsilon = 0.5$  and is located in surroundings at  $T_{surr} = 300\text{K}$ . The convective heat transfer coefficient is  $h = 10\text{ W/m}^2\text{K}$ , and the environment temperature is also  $300\text{K}$ .



(i) Determine the heat flux  $q''_b$  at  $x=L$  in  $\text{W/m}^2$ .

Solution

$$\begin{aligned} \dot{q} \cdot V &= (3 \times 10^4 \text{ W/m}^3) \cdot \frac{\pi D^2}{4} \cdot L \\ &= (3 \times 10^4 \text{ W/m}^3) \cdot \frac{\pi (0.01\text{m})^2}{4} \cdot (1\text{m}) = 2.3562\text{W} \\ &= q \end{aligned}$$

$$\dot{E}_{in} - \dot{E}_{out} + \dot{E}_{gen} = \dot{E}_{st} \rightarrow \therefore \dot{E}_{gen} = \dot{E}_{out}$$

$$\therefore q''_b = \frac{q}{A} = \frac{q}{\frac{\pi D^2}{4}} = \frac{2.3562\text{W}}{\frac{\pi (0.01\text{m})^2}{4}} = \boxed{3 \times 10^4 \text{ W/m}^2}$$

(ii) Find the effective heat transfer coefficient  $h_{rad}$  in  $W/m^2K$ . Evaluate it at an approximate surface temperature  $T_s = 400 K$ . You may use this quantity in the analysis for part (iii).

Solution

$$h_{rad} = \varepsilon \cdot \sigma \cdot (T_s + T_{Surr}) \cdot (T_s^2 + T_{Surr}^2)$$

$$T_s = 400K$$

$$T_{surr} = T_\infty = 300K \quad \varepsilon = 0.5$$

$$\begin{aligned} \therefore h_{rad} &= (0.5) \cdot (5.67 \times 10^{-8} W/m^2 K^4) \cdot (400 + 300)K \cdot (400^2 + 300^2)K^2 \\ &= \boxed{4.96125 W/m^2 K} \end{aligned}$$

(iii) Find the temperature  $T_b$  at  $x=L$  in  $^\circ K$ .

Solution

Infinite fin  $q_f = M = \sqrt{h_{tot} P k A_c} (T_b - T_\infty)$ ;  $h_{tot} = h_{rad} + h_{conv}$

$$\dot{q} \cdot V = q_t$$

$$\dot{q} \cdot A_c \cdot L = \sqrt{h_{tot} P k A_c} (T_b - T_\infty)$$

$$\therefore T_b - T_\infty = \frac{1}{\sqrt{h_{tot} P k A_c}} \cdot \dot{q} \cdot A_c \cdot L$$

$$\Rightarrow T_b = T_\infty + \frac{1}{\sqrt{h_{tot} P k A_c}} \cdot \dot{q} \cdot A_c \cdot L$$

$$A_c = \frac{\pi D^2}{4} = 0.000079m^2 \quad P = \pi D = 0.031416m$$

$$\therefore T_b = 300K + \frac{3 \times 10^4 W/m^3 \cdot (0.000079m^2) \cdot 1m}{\sqrt{(10 + 4.96125)W/m^2 K \cdot (0.031416m) \cdot (4W/mK) \cdot (0.000079m^2)}}$$

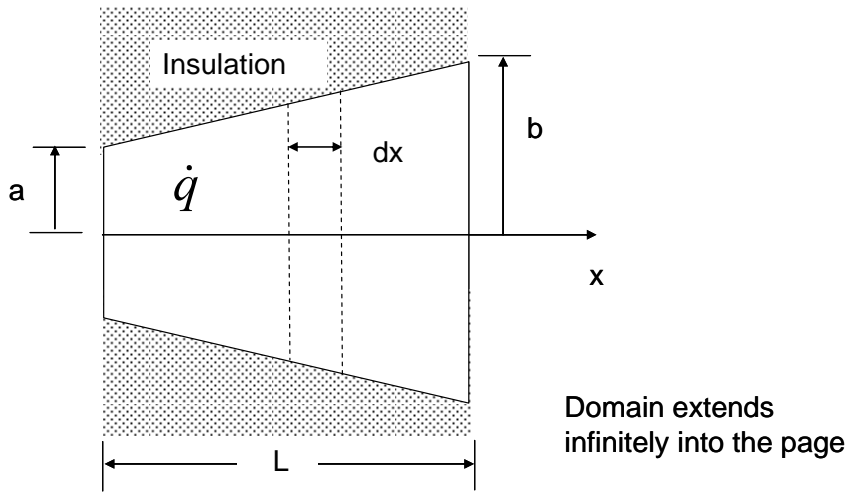
$$= 300K + \frac{2.37W}{0.012187W/K} = \boxed{494.47K}$$

(iv) Find the fin effectiveness  $\varepsilon_f$ .

Solution

$$\begin{aligned}\varepsilon_f &= \frac{q_{fin}}{h_{tot} \cdot A_{c,b} \cdot \theta_b} = \frac{\sqrt{h_{tot} P k A_c} \theta_b}{h_{tot} \cdot A_c \cdot \theta_b} = \frac{\sqrt{P k}}{\sqrt{h_{tot} \cdot A_c}} \\ &= \frac{\sqrt{(0.031416m) \cdot (4W/mK)}}{\sqrt{(14.96125W/m^2K) \cdot (0.000079m^2)}} = \boxed{10.31}\end{aligned}$$

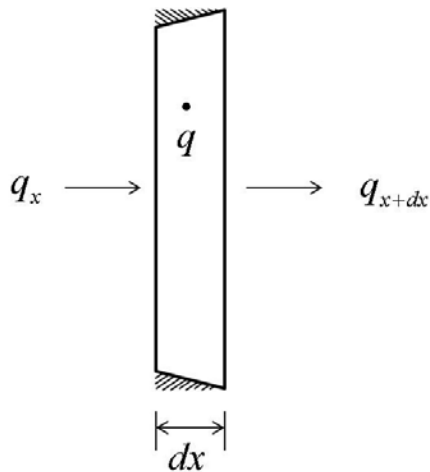
3. (40 Points) Consider steady 1D heat conduction in the trapezoidal domain shown below. The domain extends infinitely into the page. There is a constant volumetric heat generation rate  $\dot{q}$  in the domain.



The half-width of the domain at  $x=0$  is  $a$ , and that at  $x=L$  is  $b$  as shown. The temperature at  $x=0$  is  $T_0$ , while that at  $x=L$  is  $T_L$ . The thermal conductivity of the material is  $k$  and may be assumed constant.

- (i) Consider an infinitesimal control volume of extent  $dx$  as shown. Write an energy balance for the control volume to develop a differential equation for the heat transfer rate  $q_x(x)$ . State all assumptions clearly.

Solution



$$R = \frac{b-a}{L}x + a$$

$$A(x) = 2\left(a + \frac{b-a}{L}x\right)$$

$$\frac{dA(x)}{dx} = \frac{2(b-a)}{L}$$

$$\dot{E}_{in} - \dot{E}_{out} + \dot{E}_{gen} = \dot{E}_{st}$$

$$\dot{E}_{in} = q_x$$

$$\dot{E}_{out} = q_x + \frac{dq_x}{dx} dx$$

$$\dot{E}_{gen} = \dot{q} A(x) dx$$

$$q_x - q_x - \frac{dq_x}{dx} dx + \dot{q} A(x) dx = 0$$

$$-\frac{dq_x}{dx} + \dot{q} A(x) = 0$$

(ii) Now, using Fourier's law, develop a differential equation for the temperature  $T(x)$ .

Solution

$$q_x = -k \frac{dT}{dx} \cdot A(x)$$

$$\therefore \frac{d}{dx} \left( -k \frac{dT}{dx} \cdot A(x) \right) = \dot{q} A(x)$$

$$-k \frac{d}{dx} \left[ \frac{dT}{dx} \cdot A(x) \right] = -k \left[ \frac{d^2T}{dx^2} \cdot A(x) + \frac{dT}{dx} \cdot \frac{dA(x)}{dx} \right]$$

$$\therefore \frac{d^2T}{dx^2} \cdot A(x) + \frac{dT}{dx} \cdot \frac{dA(x)}{dx} = -\frac{\dot{q}}{k} A(x)$$

$$A(x) \frac{d^2T}{dx^2} + \frac{dA(x)}{dx} \frac{dT}{dx} = -\frac{\dot{q}}{k} A(x)$$

$$2 \left( a + \frac{b-a}{L} x \right) \frac{d^2T}{dx^2} + 2 \left( \frac{b-a}{L} \right) \frac{dT}{dx} + 2 \left( a + \frac{b-a}{L} x \right) \frac{\dot{q}}{k} = 0$$

(iii) Write down the boundary conditions necessary to solve the  $T(x)$  differential equation.  
DO NOT SOLVE THE EQUATION.

Solution

$$T(x=0) = T_0$$

$$T(x=L) = T_L$$